

TECNICA DELLE COSTRUZIONI

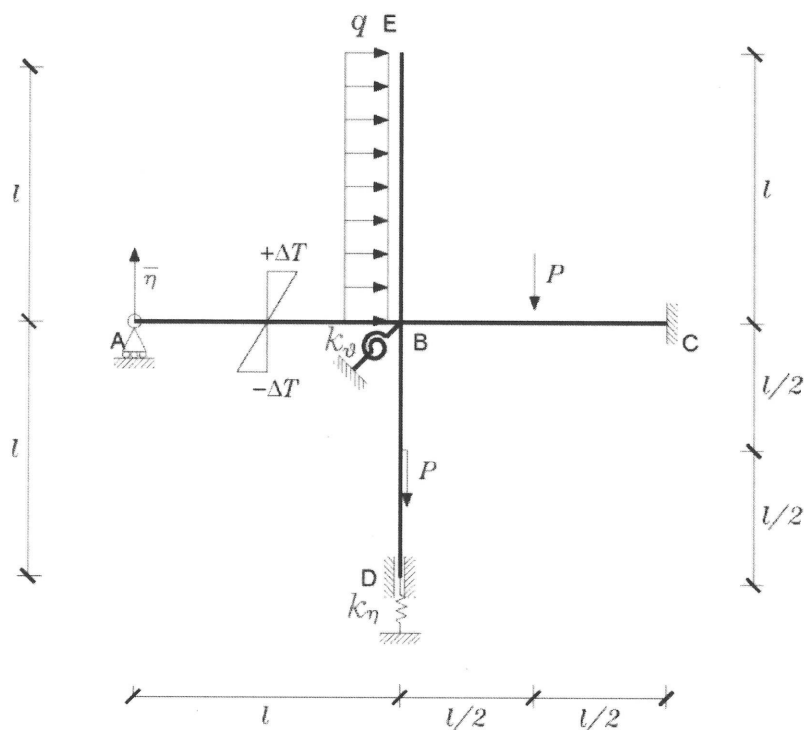
TEMA ESAME DEL 26 MARZO 2018

DOCENTE: PROF. FAUSTO MINELLI

ESERCITATORE: ING. LUCA FACCONI

DURATA: 2 ORE.

Esercizio



$$K_{\eta} = 24 \frac{EJ}{l^3}$$

$$k_{\theta} = 15 \frac{EJ}{l}$$

$$\bar{\eta} = \frac{3ql^4}{2EJ}$$

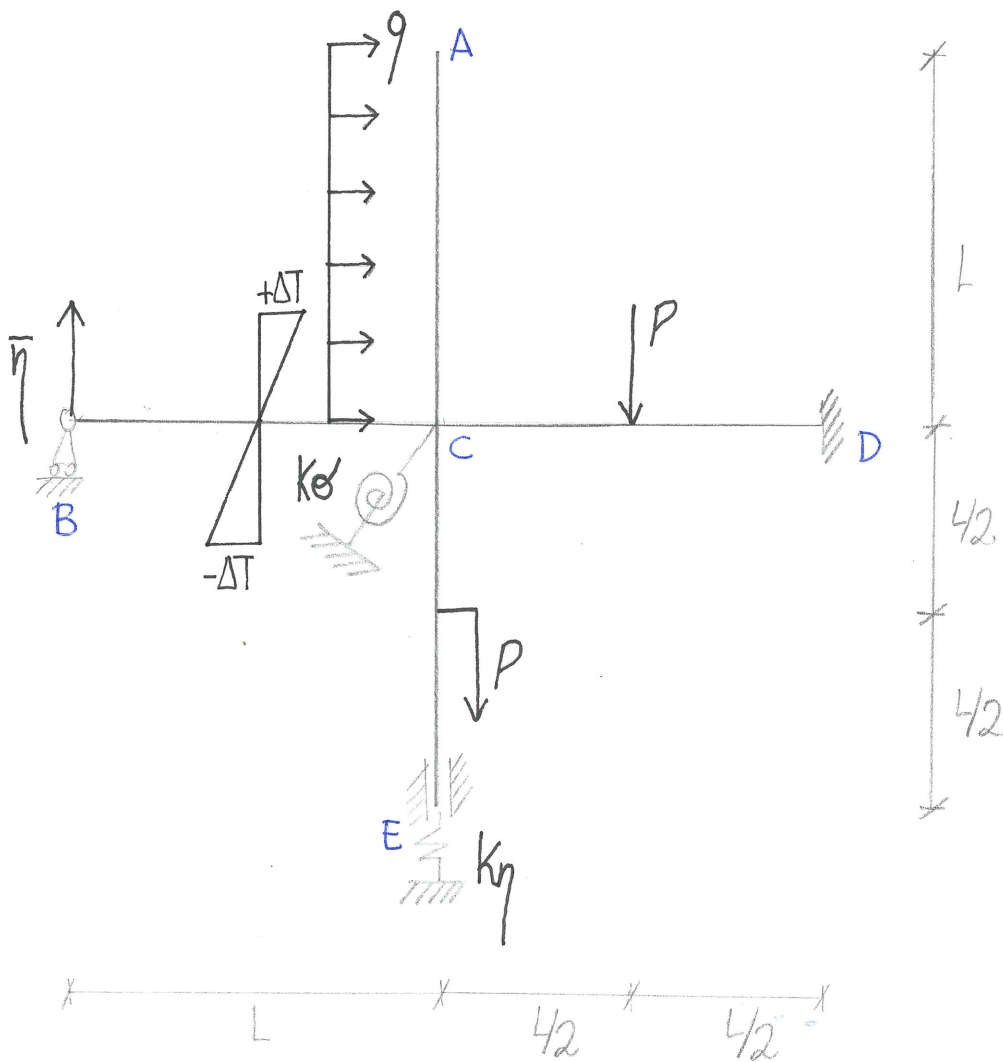
$$\frac{\alpha \Delta T}{H} = \frac{1}{3} \frac{ql^2}{EJ}$$

$$P = 2ql$$

Dato il telaio in figura, si richiedono i grafici di:

1. Momento flettente (con il valore e la posizione dei massimi);
2. Taglio;
3. Azione assiale;
4. Deformata qualitativa con posizione dei flessi.

TEMA ESAME 26/03/2018



DATI

$$\bar{\eta} = \frac{3}{2} \frac{qL^4}{EJ}$$

$$\frac{\Delta T}{H} = \frac{qL^2}{3EJ}$$

$$K\theta = 15 \frac{EJ}{L}$$

$$K\eta = 24 \frac{EJ}{L}$$

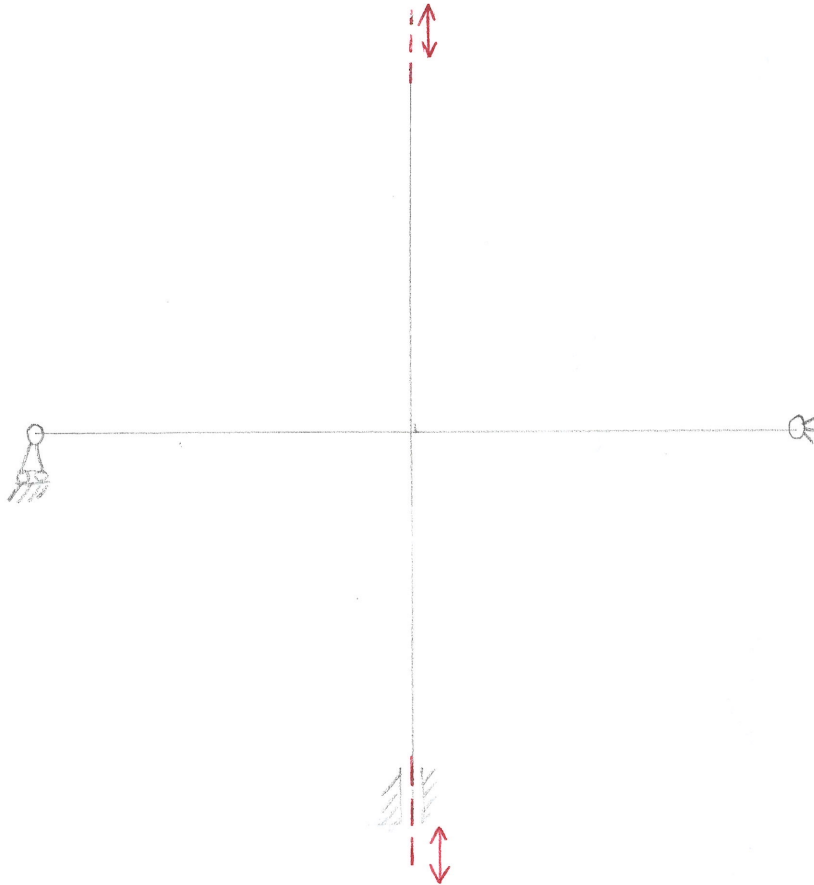
$$P = 2qL$$

SI VALUTA IL GRADO DI IPERSTATICITA' DELLA STRUTTURA

$$GdV = 8$$

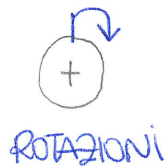
$$GdL = 3$$

SI VALUTA SE IL TELAIIO È A NODI FISSI O SPOSTABILI

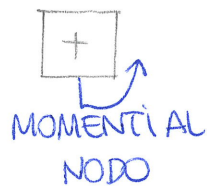


SI OSSERVA CHE È POSSIBILE UN CINEMATISMO IN DIREZIONE VERTICALE
IL TELAIIO È A NODI SPOSTABILI → BIELLA FITIZIA IN A

IN CONVENZIONE USATE PER LA RISOLUZIONE DEL TELAIIO



ROTAZIONI



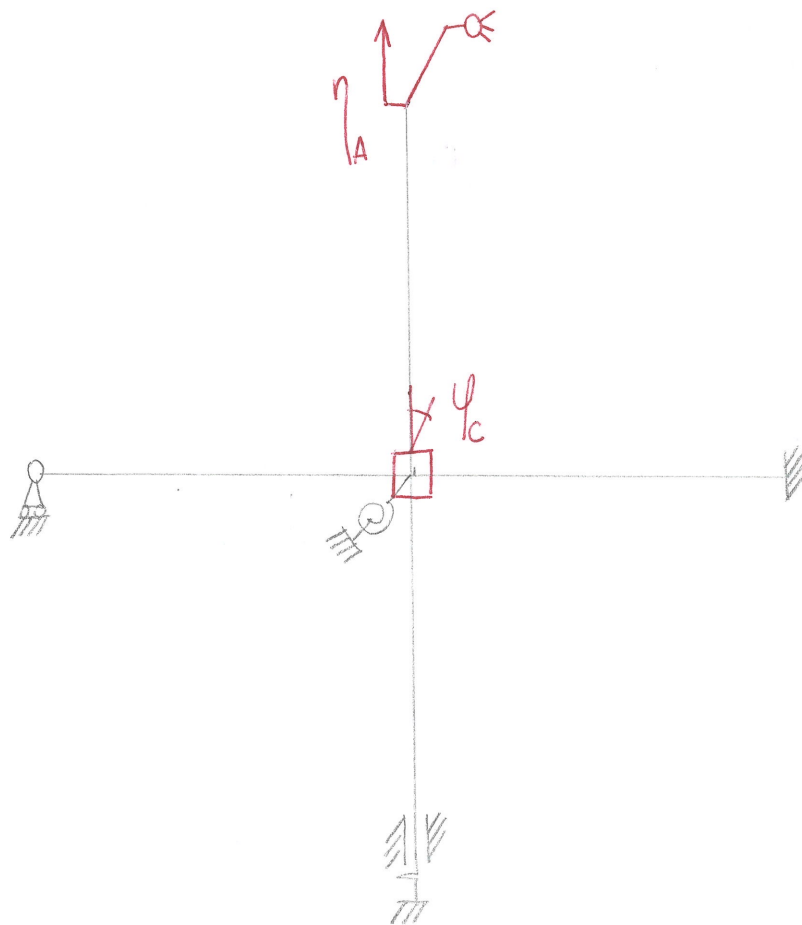
MOMENTI AL
NODO



SI RISOLVE CON IL METODO DEGLI SPOSTAMENTI

LE INCOGNITE SONO:

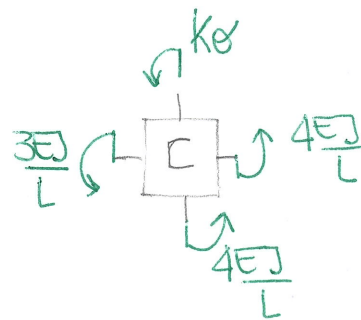
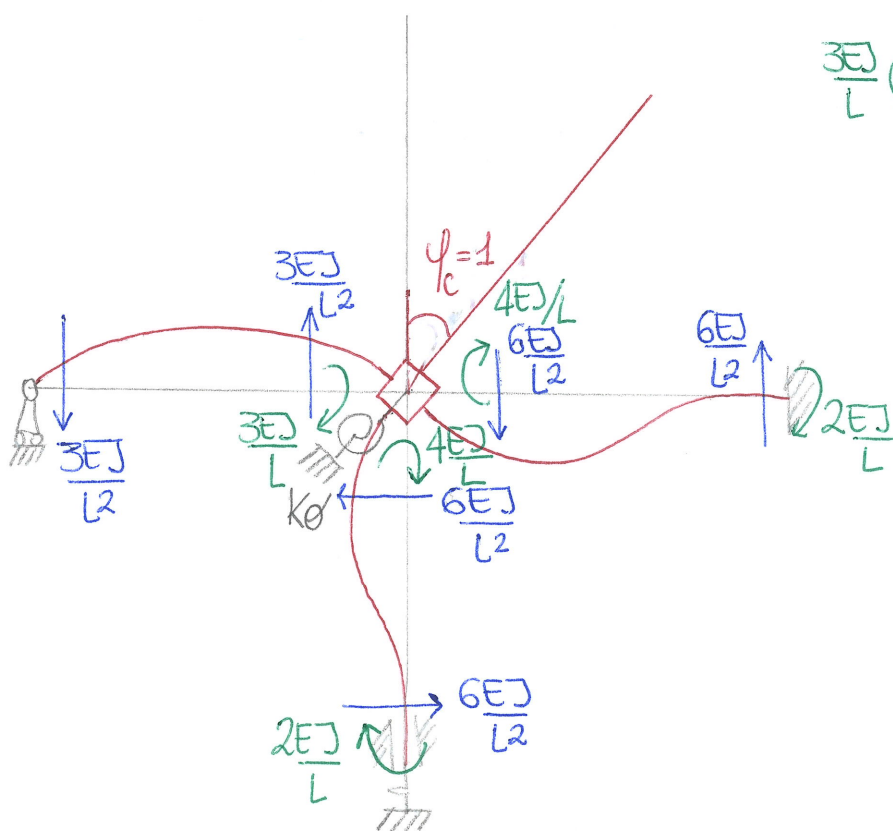
$$\begin{cases} \varphi_c \\ \eta_A \end{cases}$$



IL SISTEMA RISOLVENTE RISULTA ESSERE:

$$\begin{cases} \sum M_C = 0 \\ \sum R_A = 0 \end{cases} \quad \begin{cases} M_{CC} \psi_c + M_{CA} \eta_A + M_{CO} = 0 \\ h_{AC} \psi_c + h_{AA} \eta_A + h_{AO} = 0 \end{cases}$$

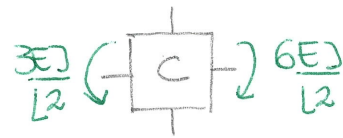
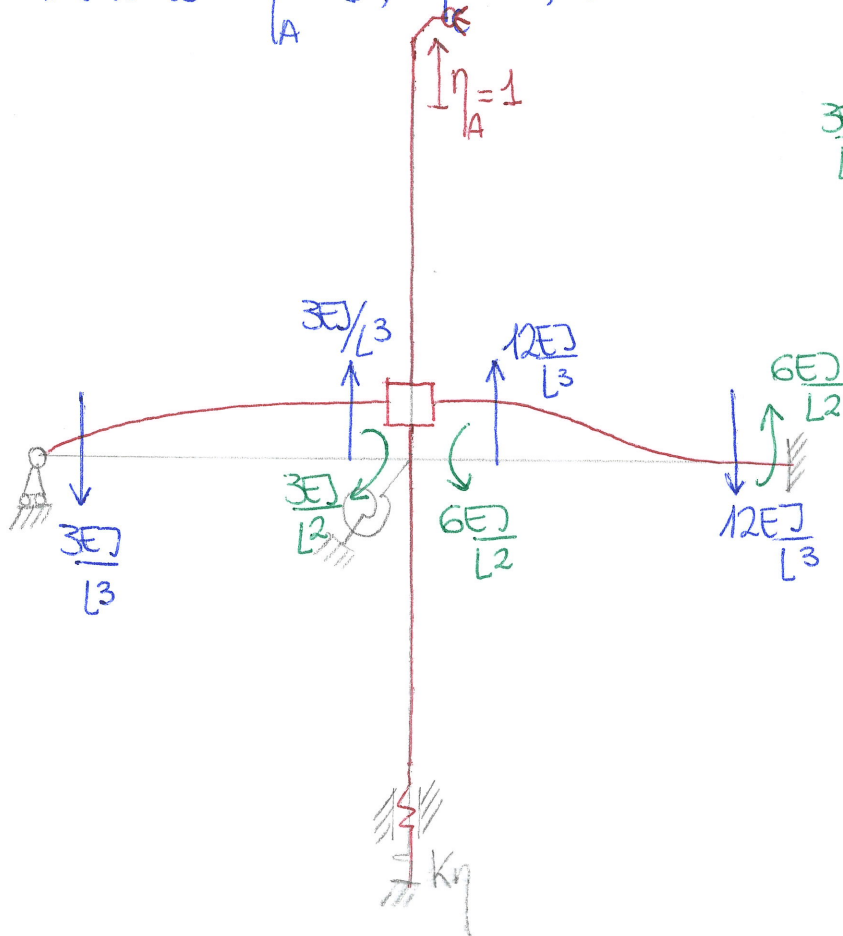
*CASO 1: $\psi_c = 1$; $\eta_A = 0$; $\forall R_{EX} = 0$



$$\begin{aligned} M_{CC} &= \frac{3EJ}{L} + \frac{4EJ}{L} + \frac{4EJ}{L} + \frac{15EJ}{L} = \\ &= 26 \frac{EJ}{L} \end{aligned}$$

$$h_{AC} = \frac{3EJ}{L^2} - \frac{6EJ}{L^2} = -\frac{3EJ}{L^2}$$

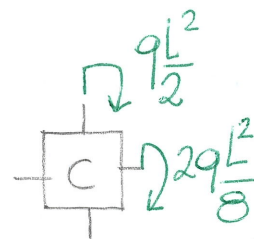
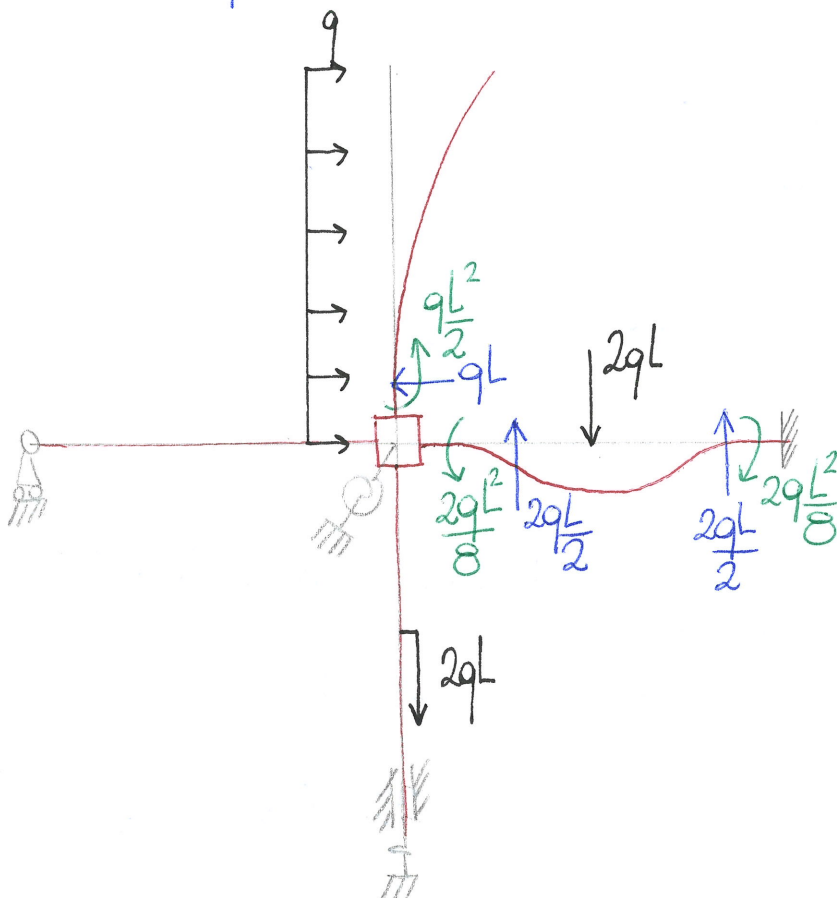
* CASO 2: $\eta_A = 1$; $\psi = 0$; $\forall R_x = 0$



$$M_{CA} = \frac{3EJ}{L^2} - \frac{6EJ}{L^2} = -\frac{3EJ}{L^2}$$

$$h_{AA} = \frac{3EJ}{L^3} + \frac{12EJ}{L^3} + \frac{24EJ}{L^3} = \frac{39EJ}{L^3}$$

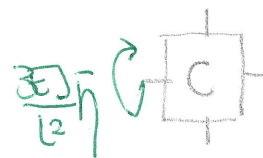
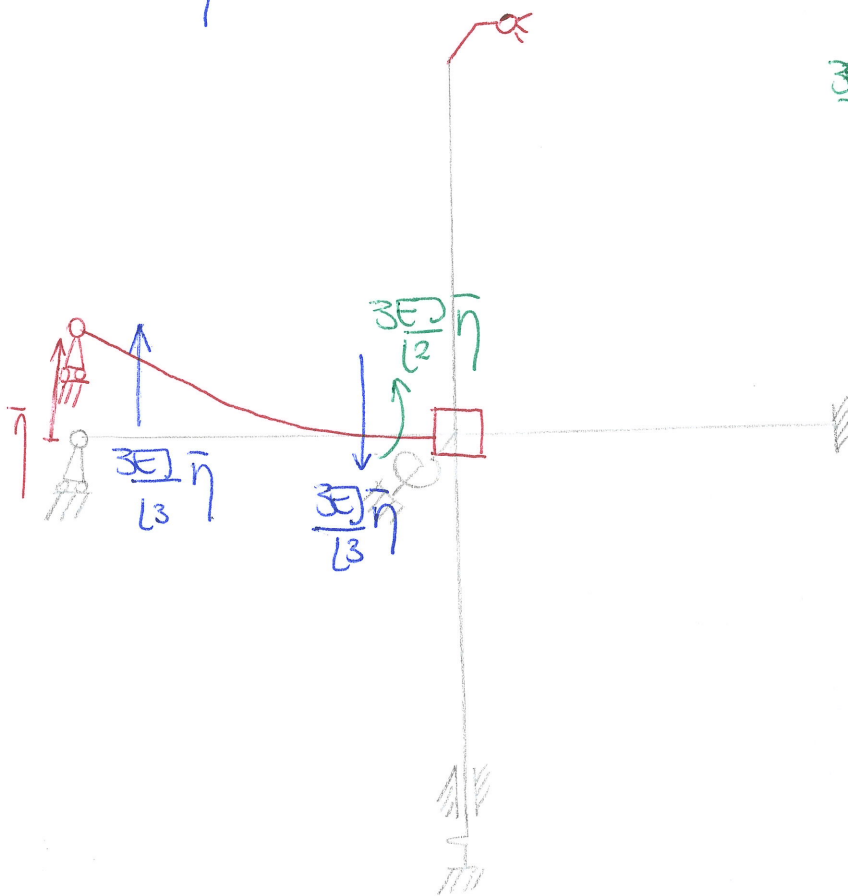
* CASO 3: $q \neq 0$



$$m_{C0,1} = -\frac{qL^2}{2} - \frac{2qL^2}{8} = -\frac{3}{4}qL^2$$

$$h_{A0,1} = 2qL + \frac{2qL}{2} = 3qL$$

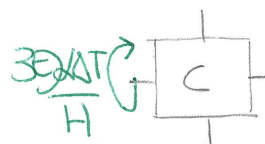
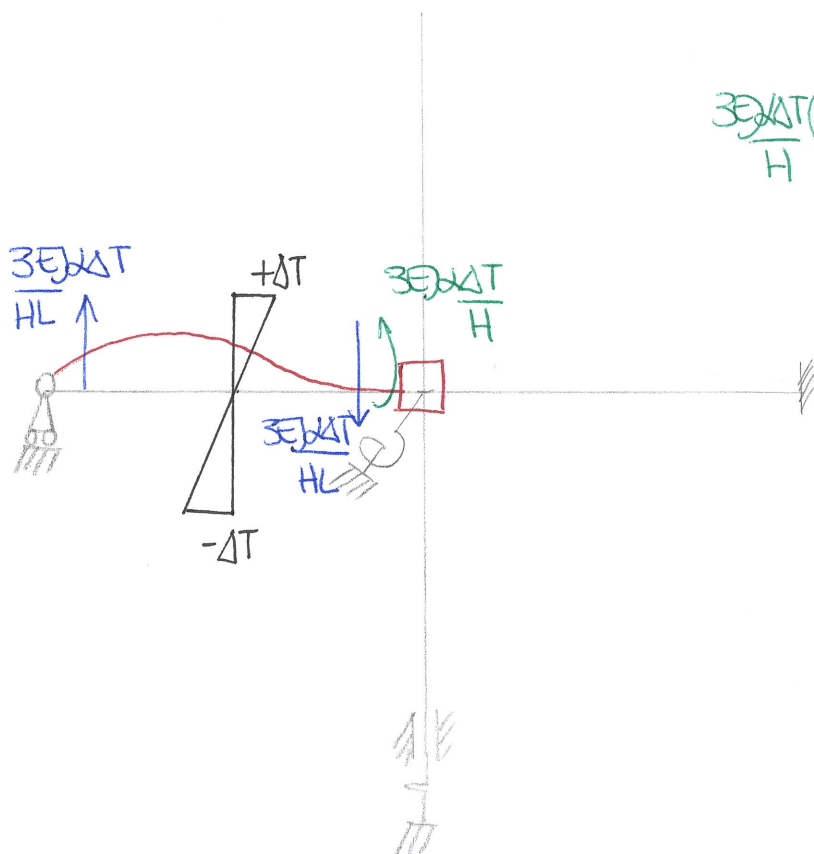
* CASO 4 $\bar{\eta} \neq 0$



$$m_{co,2} = -\frac{3EI}{L^2} \cdot \bar{\eta} = -\frac{9}{2} q l^2$$

$$h_{AO,2} = -\frac{3EI}{L^3} \bar{\eta} = -\frac{9}{2} q l^2$$

* CASO 5: $\Delta T \neq 0$



$$m_{co,3} = -\frac{3EI\alpha\Delta T}{H} = -q l^2$$

$$h_{AO,3} = -\frac{3EI\alpha\Delta T}{HL} = -q l$$

$$M_{BO} = -\frac{3}{4} q l^2 - \frac{9}{2} q l^2 - q l^2 = -\frac{25}{4} q l^2$$

$$h_{AO} = 3qL - \frac{9}{2} q l^2 - qL = -\frac{5}{2} qL$$

IL SISTEMA RISOLVENTE

$$\begin{cases} 26 \frac{EJ}{L} \cdot \varphi_c - \frac{3EJ}{L^2} \cdot \eta_A - \frac{25}{4} q l^2 = 0 & (1) \end{cases}$$

$$\begin{cases} -3 \frac{EJ}{L^2} \cdot \varphi_c + \frac{39EJ}{L^3} \cdot \eta_A - \frac{5}{2} qL = 0 & (2) \end{cases}$$

MOLTIPLICO (2) PER $\frac{L}{13}$ E LA SOMMO A (1)

$$-\frac{3}{13} \frac{EJ}{L} \varphi_c + 3 \frac{EJ}{L^2} \eta_A - \frac{5}{26} q l^2 = 0$$

(+)

$$26 \frac{EJ}{L} \varphi_c - 3 \frac{EJ}{L^2} \eta_A - \frac{25}{4} q l^2 = 0$$

$$\left[-\frac{3}{13} \frac{EJ}{L} + 26 \frac{EJ}{L} \right] \varphi_c + \left[3 \frac{EJ}{L^2} - 3 \frac{EJ}{L^2} \right] \eta_A + \left[-\frac{5}{26} q l^2 - \frac{25}{4} q l^2 \right] = 0$$

$$\frac{335}{13} \frac{EJ}{L} \varphi_c - \frac{335}{52} q l^2 = 0$$

$$\varphi_c = \frac{335}{52} q l^2 \cdot \frac{13}{335} \frac{L}{EJ} \rightarrow \varphi_c = \frac{1}{4} \frac{q l^3}{EJ}$$

SOSTITUISCO φ_c NELLA (2)

$$-3 \frac{EJ}{L^2} \cdot \frac{1}{4} \frac{q l^3}{EJ} + \frac{39EJ}{L^3} \eta_A - \frac{5}{2} qL = 0$$

$$\frac{39EJ}{L^3} \eta_A - \frac{3}{4} qL - \frac{5}{2} qL = 0$$

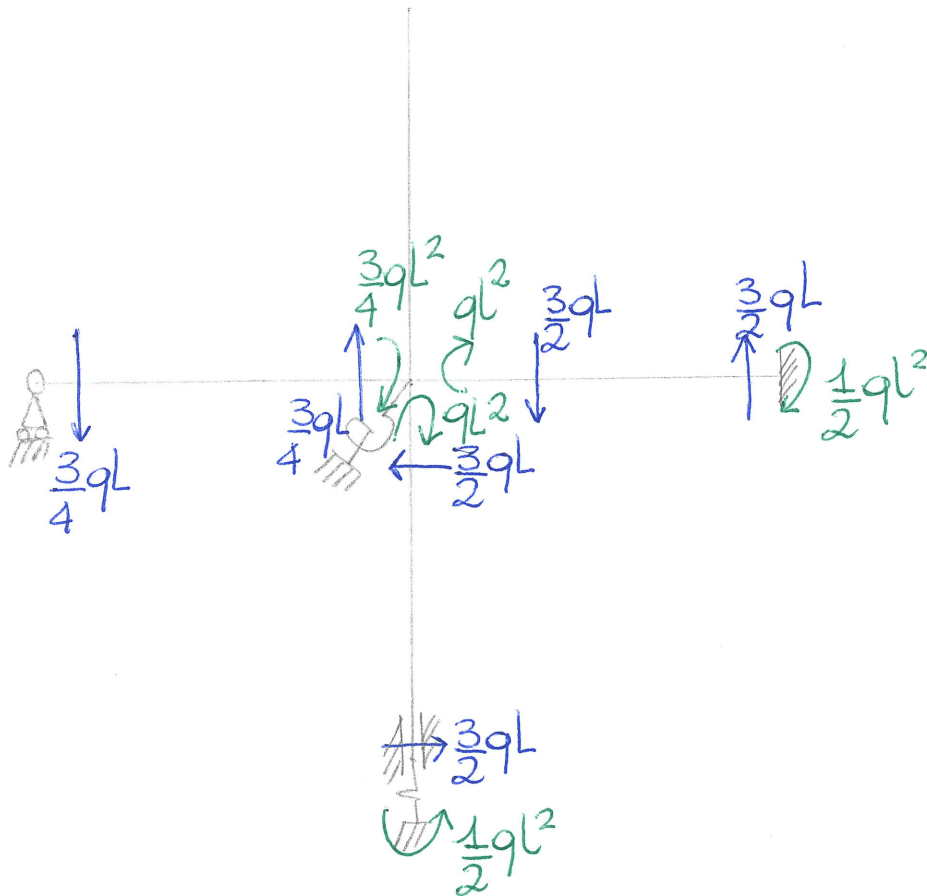
$$\frac{39EJ}{L^3} \eta_A = \frac{13}{4} qL$$

$$\eta_A = \frac{13}{4} qL \cdot \frac{1}{39} \frac{L^3}{EJ} \rightarrow \eta_A = \frac{1}{12} \frac{qL^4}{EJ}$$

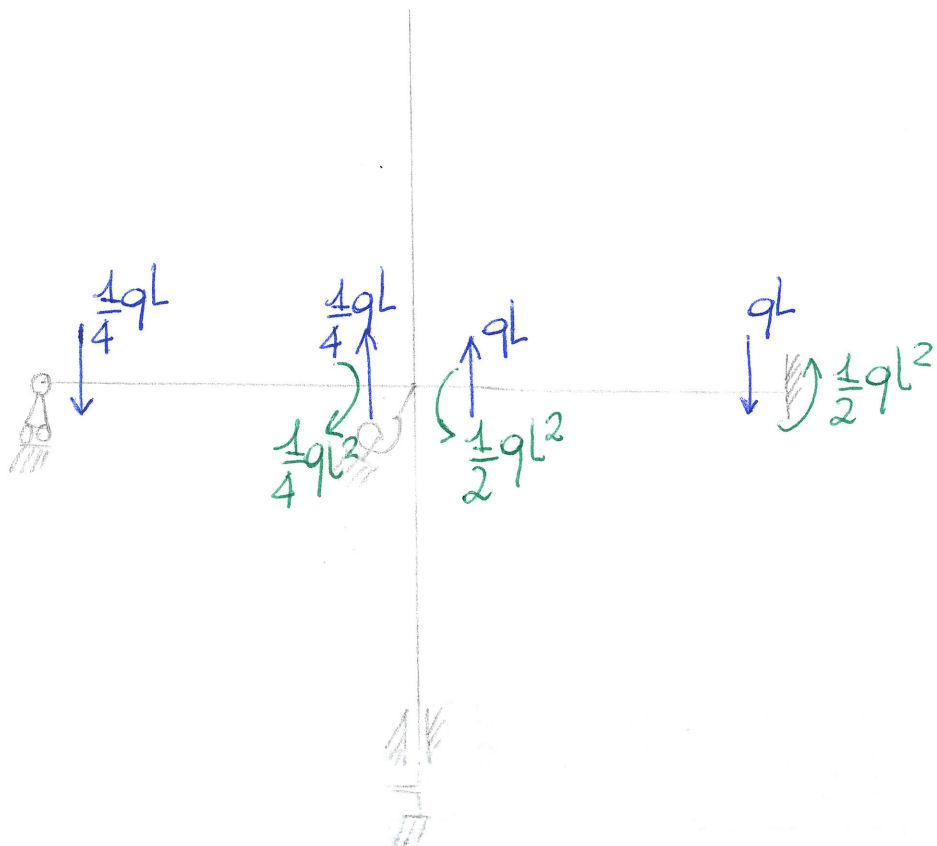
IN CONCLUSIONE

$$\begin{cases} \psi_c = \frac{1}{4} \frac{qL^3}{EI} \\ \eta_A = \frac{1}{12} \frac{qL^4}{EI} \end{cases}$$

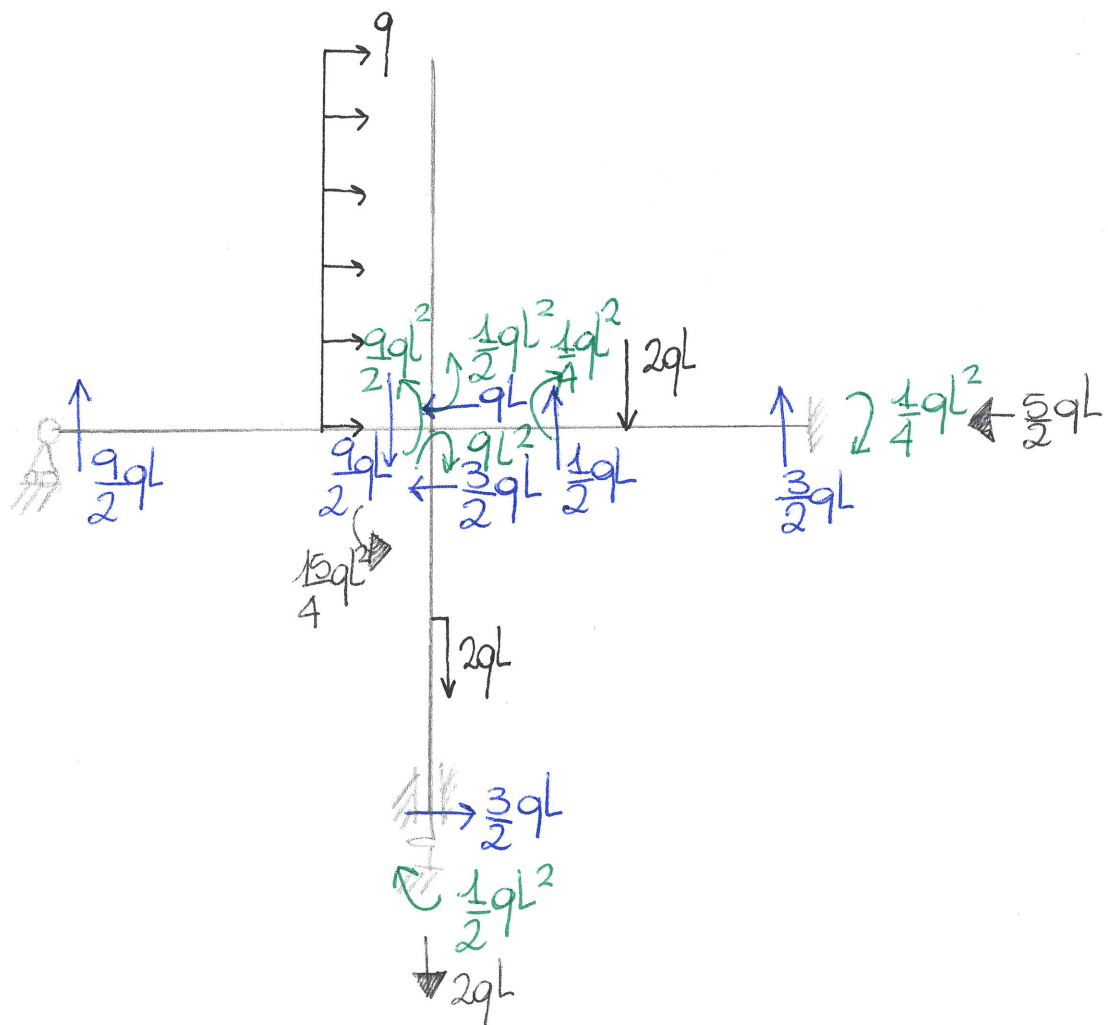
$$\approx \psi_c = \frac{1}{4} \frac{qL^3}{EI}$$



$$\eta_A = \frac{1}{12} \frac{qL^4}{EI}$$



~ CALCOLO AZIONI INTERNE



$$EQ: \uparrow + = 0$$

$$\frac{9}{2}qL + \frac{3}{2}qL - 2qL - 2qL - 2qL = 0 \quad ok!$$

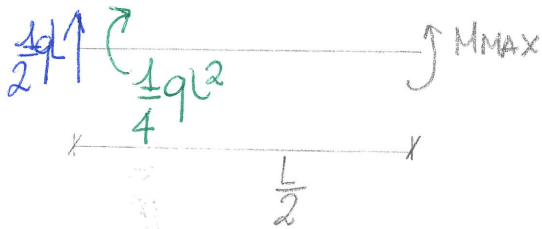
$$EQ: \rightarrow = 0$$

$$\frac{3}{2}qL + qL - \frac{5}{2}qL = 0 \quad ok!$$

$$EQ: \uparrow_B = 0$$

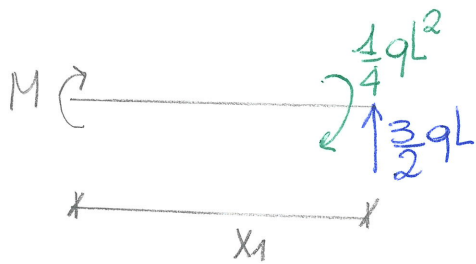
$$-\frac{3}{2}qL \cdot L + \frac{1}{2}qL^2 + 2qLL + 2qLL - \frac{3}{2}qL \cdot 2L + \frac{1}{4}qL^2 + 2qL \cdot \frac{3}{2}L + qL \cdot \frac{L}{2} - \frac{15}{4}qL^2 = 0 \quad ok!$$

N MOMENTO MASSIMO



$$M_{max} = \frac{1}{4}qL^2 + \frac{1}{2}qL \cdot \frac{L}{2} = \frac{1}{2}qL^2$$

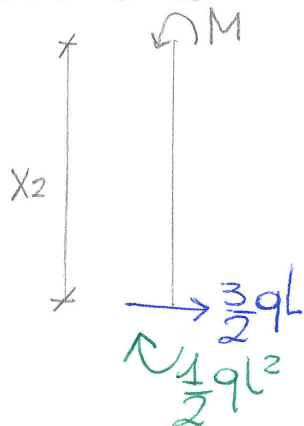
N FLESSO 1



$$M = \frac{1}{4}qL^2 - \frac{3}{2}qL \cdot x_1 = 0$$

$$\frac{3}{2}x_1 = \frac{1}{4}L \rightarrow x_1 = \frac{2}{3} \cdot \frac{1}{4}L = \frac{1}{6}L \approx 0,17L$$

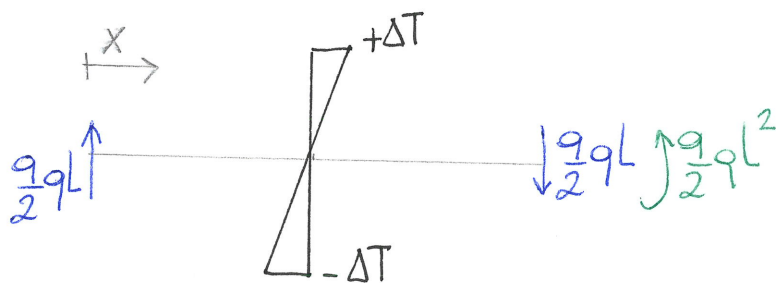
N FLESSO 2



$$M = -\frac{3q}{2}x_2 + \frac{1}{2}qL^2 = 0$$

$$\frac{3}{2}x_2 = \frac{1}{2}L \rightarrow x_2 = \frac{2}{3} \cdot \frac{L}{2} = \frac{L}{3} \approx 0,33L$$

N DEFORMATA TERMICA



$$y''(x) = -\frac{M(x)}{EJ} + \frac{2\Delta T}{EJ}$$

$$M(x) = \frac{q}{2}qL \cdot x$$

$$y''(x) = -\frac{q}{2}qL \cdot \frac{x}{EJ} + 2 \cdot \frac{1}{3} \frac{qL^2}{EJ}$$

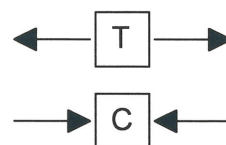
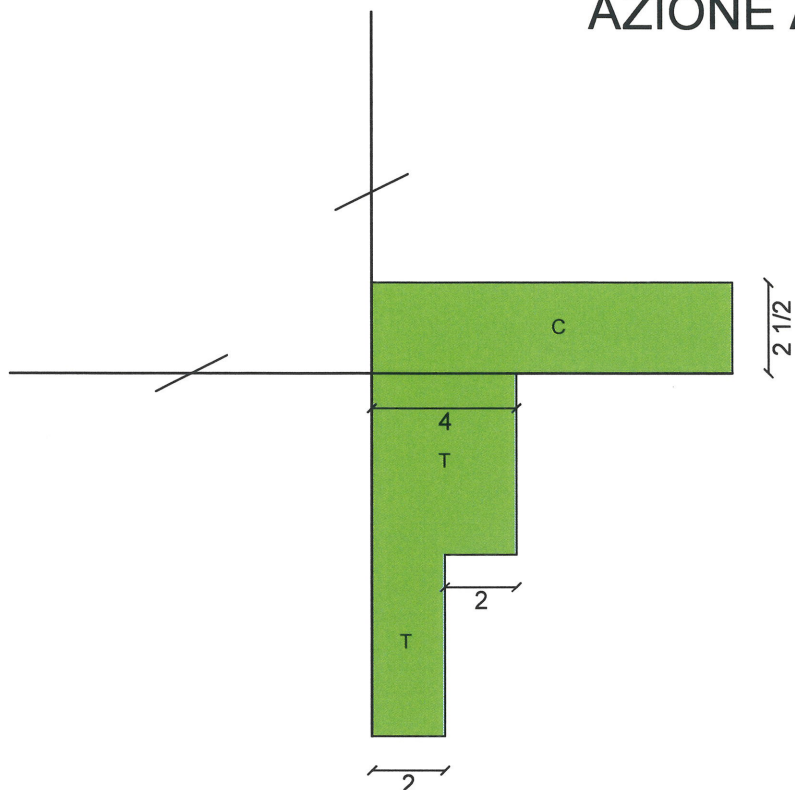
$$y''(x) \cdot EJ = -\frac{q}{2}qLx + \frac{2}{3}qL^2 \geq 0$$

$$\frac{q}{2}x \leq \frac{2}{3}L \rightarrow x \leq \frac{2}{q} \cdot \frac{2}{3}L = \frac{4}{24}L \approx 0,15L$$

$$x \leq 0,15L$$

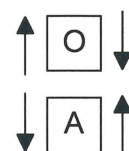
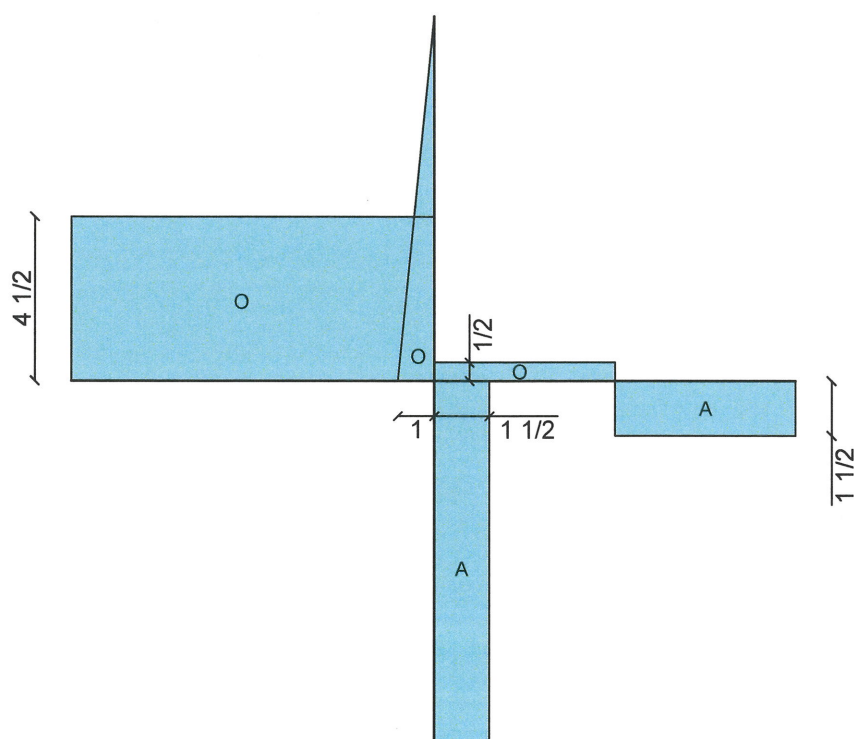
AZIONE ASSIALE

U.D.M. ql



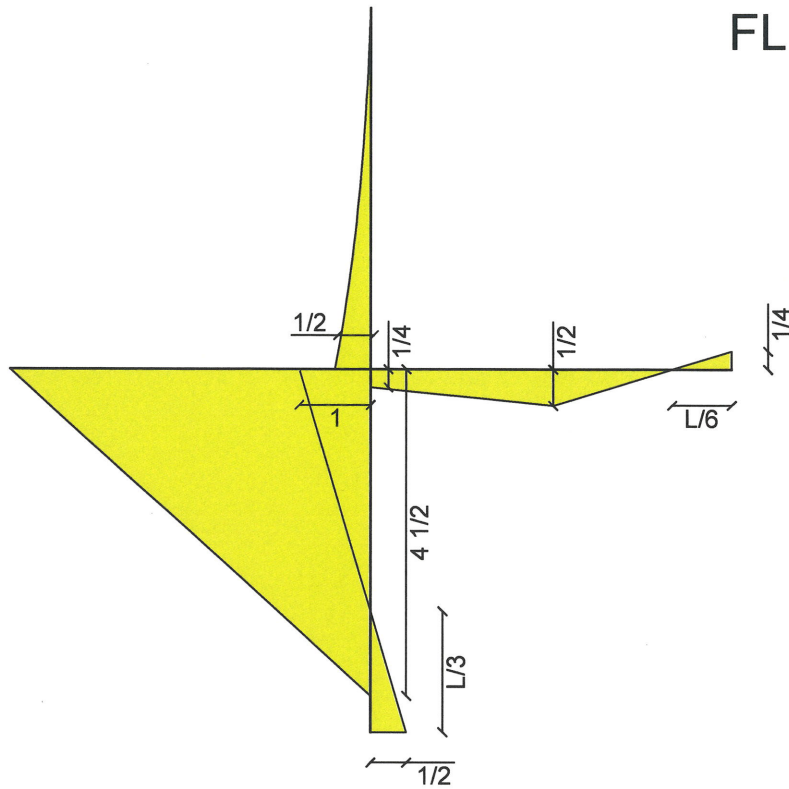
TAGLIO

U.D.M. ql



U.D.M. ql^2

U.D.M. ql^2



DEFORMATA QUALITATIVA

U.D.M. Spostamenti ql^4/EJ U.D.M. Rotazioni qI^3/EJ 