

STRUCTURAL DESIGN

TEST 2022/09/05

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Consider the frame shown in Figure 1.

- 1- Plot the diagrams of the frame internal actions (N,M,V);
- 2- Make a qualitative sketch of the frame's deflected shape and evidence the inflection points.

Assume for each member of the frame: $EA = \infty$, $EJ = \text{constant}$.

Furthermore, assume:

$$k_{\eta} = \frac{35 EJ}{8 l^3} \quad k_{\vartheta} = \frac{4 EJ}{5 l} \quad \bar{\vartheta} = \frac{1 ql^3}{8 EJ} \quad P = 4q \cdot l$$

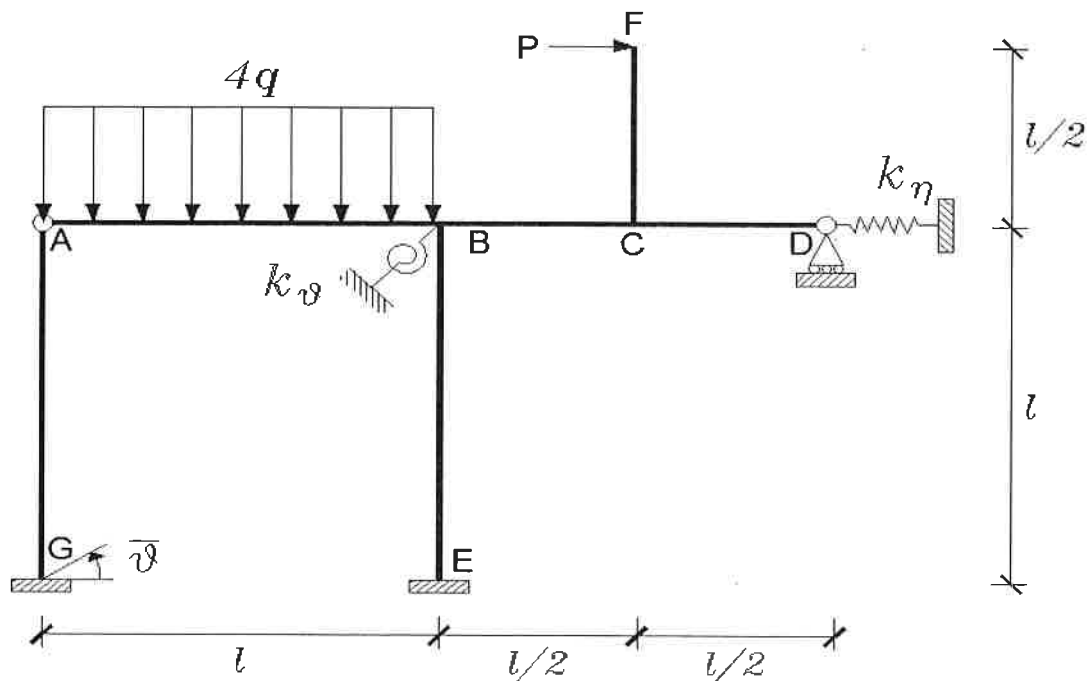


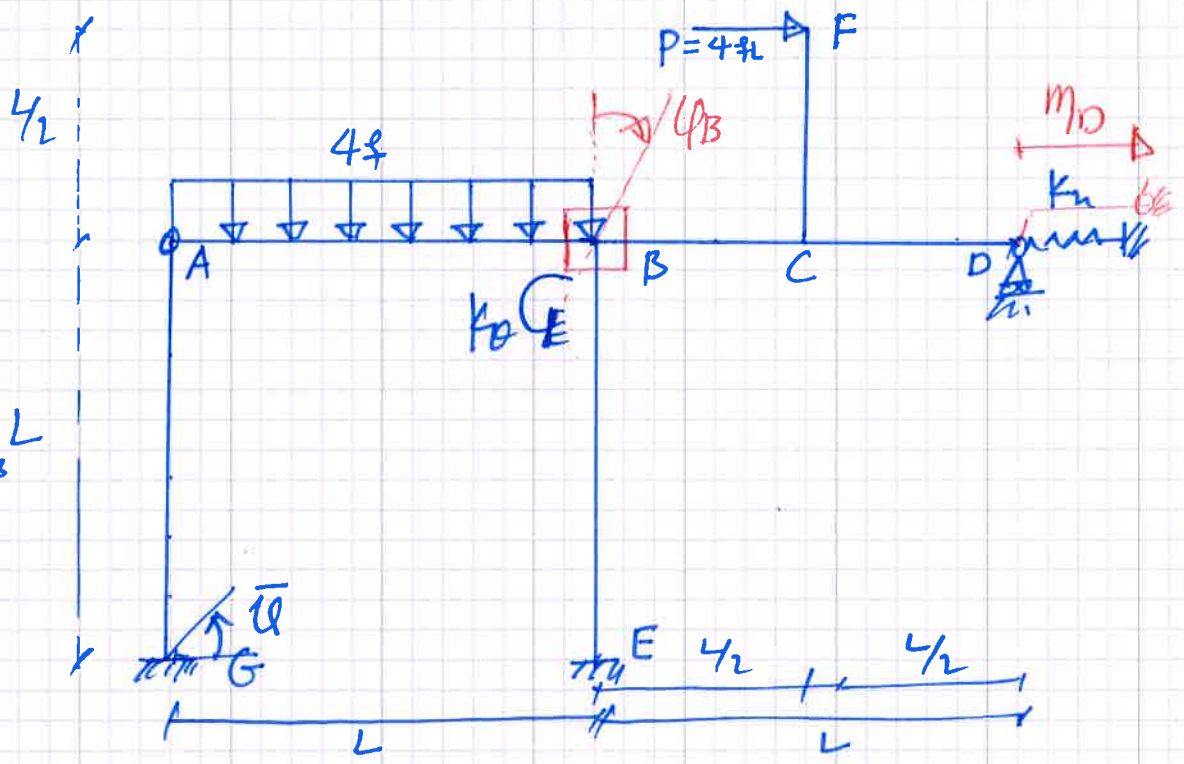
Figure 1

Given data

$$k_{\theta} = \frac{4}{5} \frac{EI}{L}$$

$$k_{\eta} = \frac{35}{8} \frac{EI}{L^3}$$

$$k_{\xi} = \frac{1}{8} \frac{9EI}{EI}$$

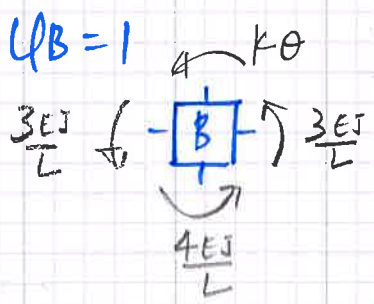


$$M_{BB} \phi_B + M_{BD} m_D + M_B \phi = 0$$

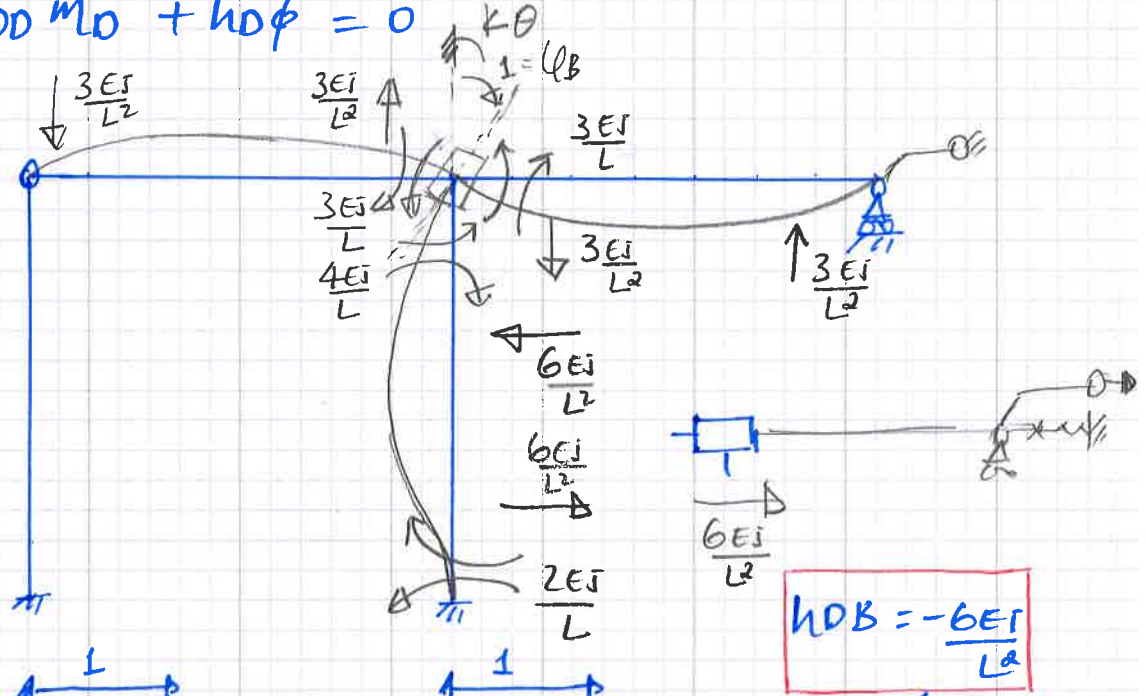
$$k_{\theta B} \phi_B + k_{\eta D} m_D + k_{\phi} \phi = 0$$

Convention

Solution

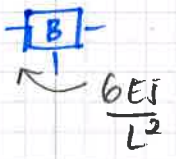


$$M_{BB} = \frac{10EI}{L} + k_{\theta}$$

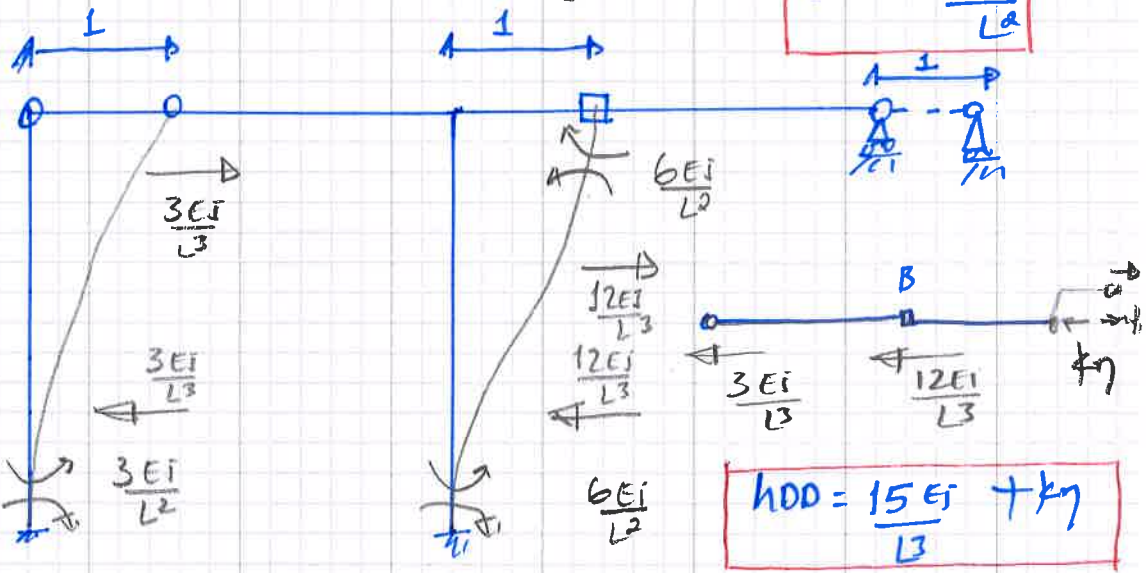


$$M_{DB} = -\frac{6EI}{L}$$

$m_D = 1$

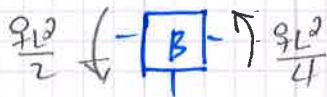
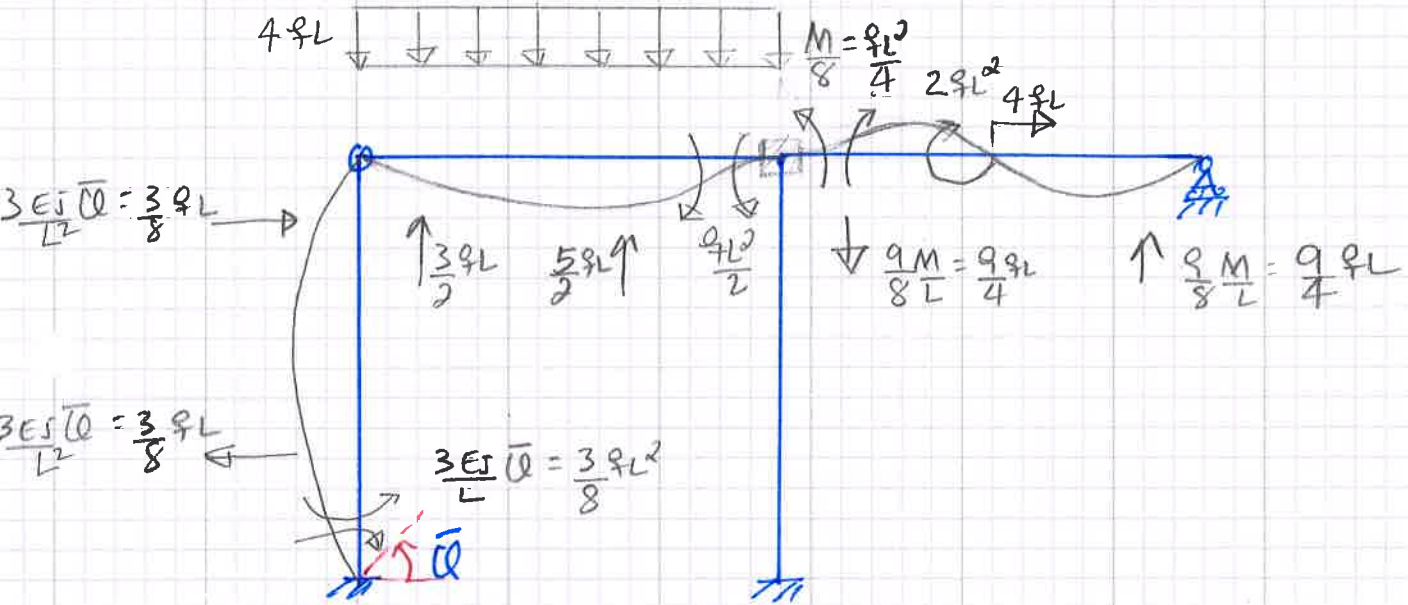
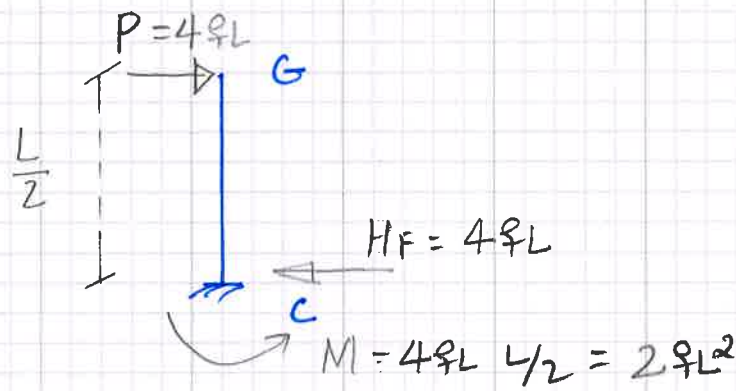


$$M_{BD} = -\frac{6EI}{L}$$

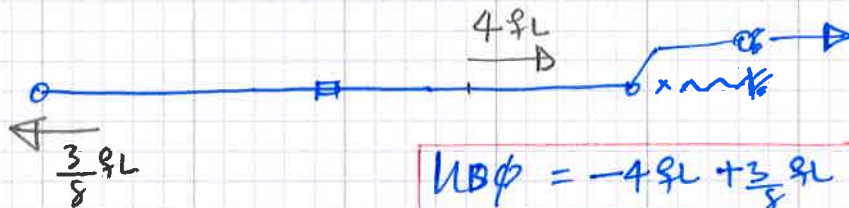


$$k_{\eta D} = \frac{15EI}{L^3}$$

$\varphi \neq 0$



$$M_{B\phi} = \frac{3}{4} \phi L^2$$



$$M_{B\phi} = -4\phi L + \frac{3}{8} \phi L^2 = -\frac{29}{8} \phi L$$

$$\begin{cases} M_{BB} \varphi_B + M_{BD} m_D + M_{B\phi} = 0 \\ h_{DB} \varphi_B + h_{DD} m_D + h_{D\phi} = 0 \end{cases}$$

$$\left(\frac{10EI}{L} + k_0 \right) \varphi_B - \frac{6EI}{L^2} m_D + \frac{3}{4} \phi L^2 = 0$$

$$-\frac{6EI}{L^2} \varphi_B + \left(\frac{15EI}{L^3} + k_m \right) m_D - \frac{29}{8} \phi L = 0$$

$$\frac{54}{5} \frac{EI}{L} \varphi_B - \frac{6EI}{L^2} m_D + \frac{3}{4} \phi L^2 = 0$$

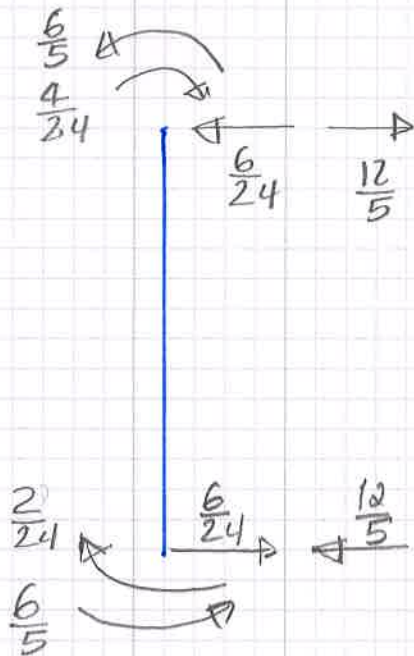
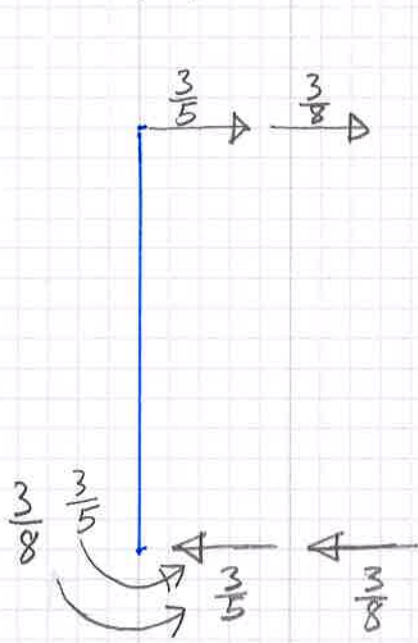
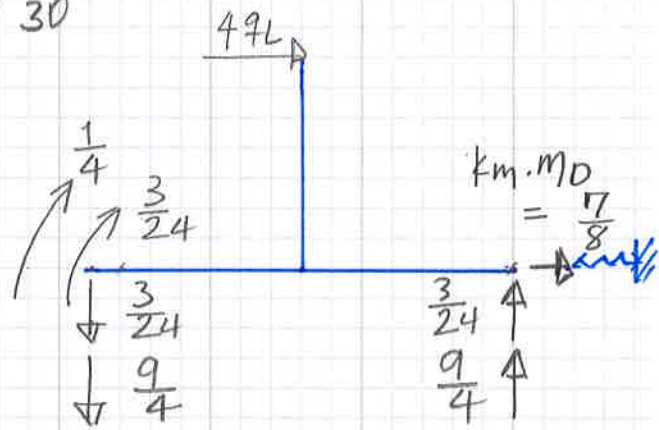
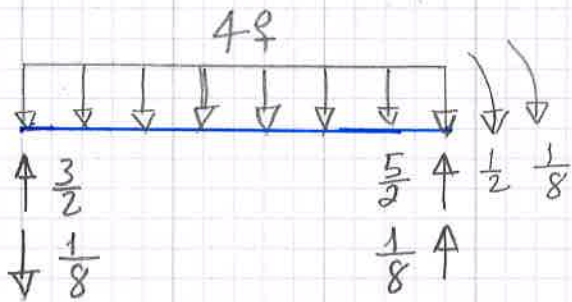
$$-\frac{6EI}{L^2} \varphi_B + \frac{155}{8} m_D - \frac{29}{8} \phi L = 0$$

$$\varphi_B = \frac{1}{24} \quad m_D = \frac{1}{5} \quad \varphi_B = \frac{1}{24} \frac{\phi L^3}{EI}, \quad m_D = \frac{1}{5} \frac{\phi L^4}{EI}$$

Internal actions

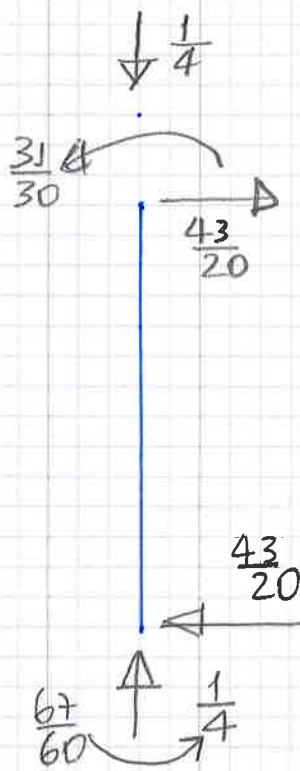
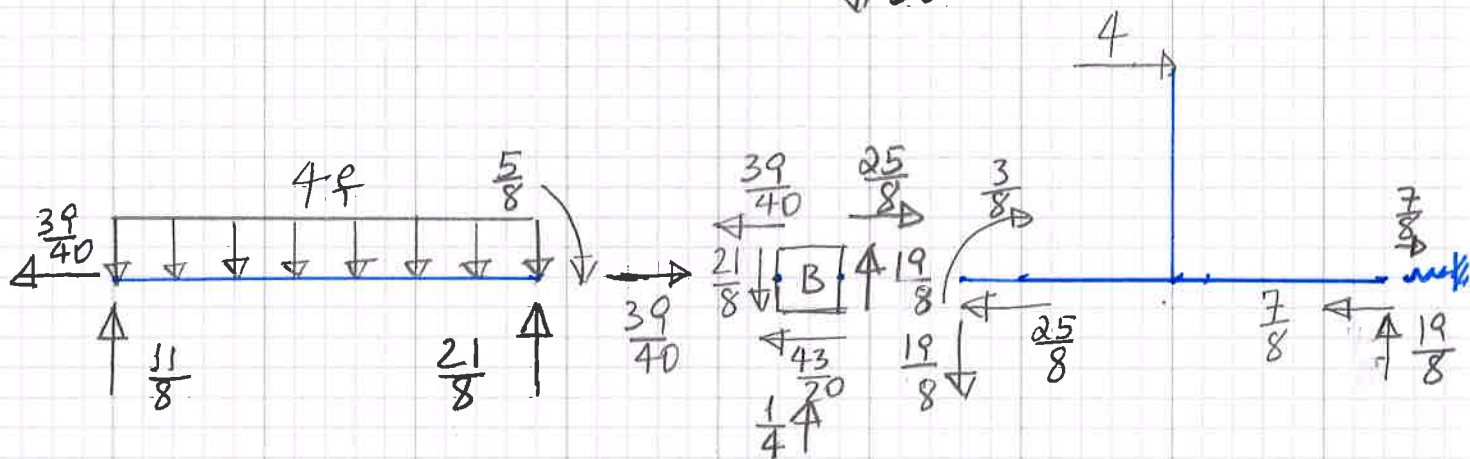
$$k_0 = \frac{1}{24} \times \frac{4}{5}$$

$$\downarrow \frac{1}{30}$$

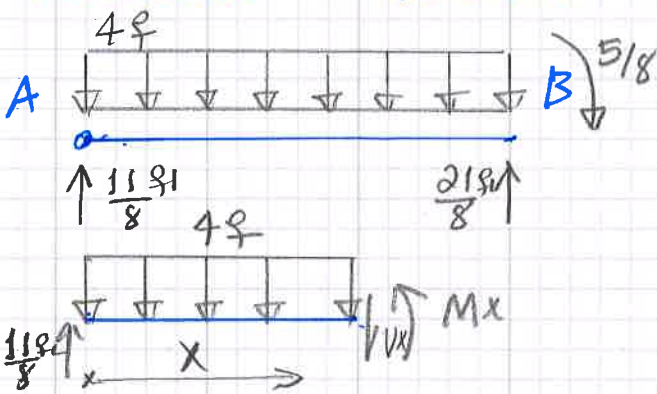


Resultant internal actions

$$\curvearrowright \frac{1}{30}$$



The maximum bending moment point on member AB



$$Mx = \frac{11}{8} qLx - \frac{4}{2} qx^2$$

$$Mx = \frac{11}{8} qLx - \frac{4}{2} qx^2 = 0$$

$$\frac{11}{8} qLx - \frac{4}{2} qx^2 = 0$$

$$x_2 = \frac{11}{16}$$

$$Mx + \frac{4qx^2}{2} - \frac{11qLx}{8} = 0$$

$$Mx = \frac{11}{8} qLx - \frac{4}{2} qx^2$$

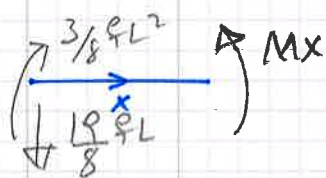
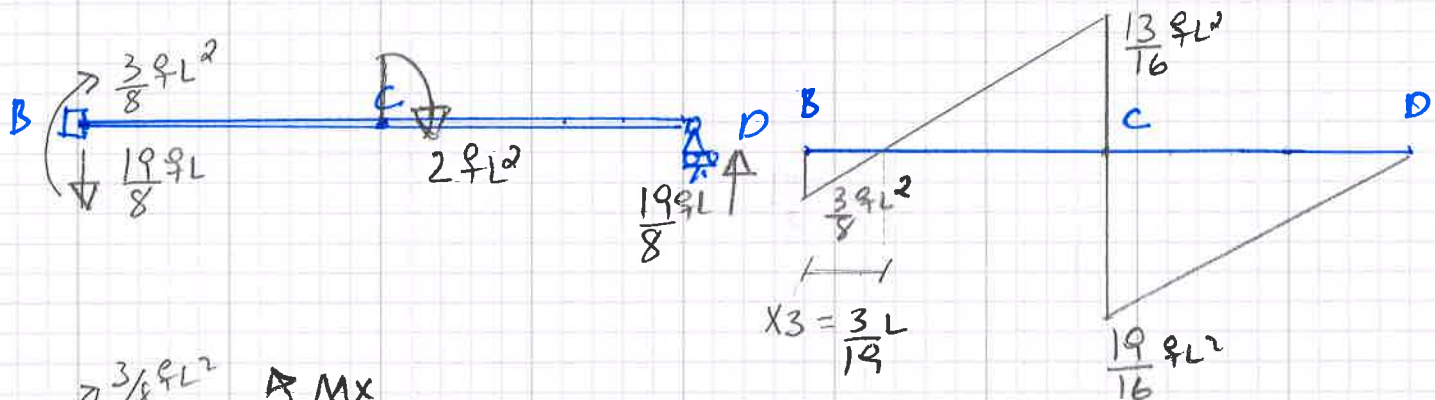
$$M'(x) = \frac{11}{8} qL - \frac{8}{2} qx = 0$$

$$x_1 = \frac{11}{32} L$$

$$M_{\max} \left(\frac{11}{32} \right) = \frac{11}{8} qL \left(\frac{11}{32} \right) - 4q \left(\frac{11}{32} \right)^2$$

$$M_{\max} = \frac{121}{512}$$

Inflection points on members BD and BE

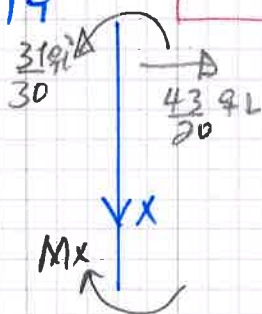
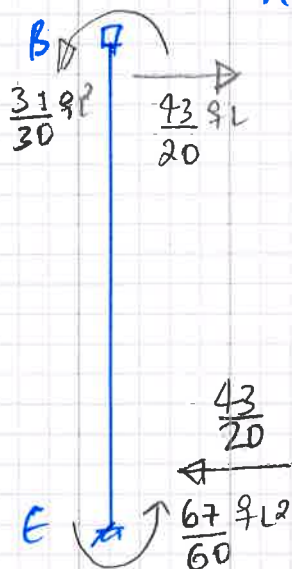


$$M_x = \frac{3}{8} qL^2 - \frac{19}{8} qLX = 0$$

$$\frac{3}{8} qL^2 - \frac{19}{8} qLX = 0$$

$$\frac{3}{8} qL^2 = \frac{19}{8} qLX$$

$$X = \frac{3}{19} L \Rightarrow X_3 = \frac{3}{19} L$$

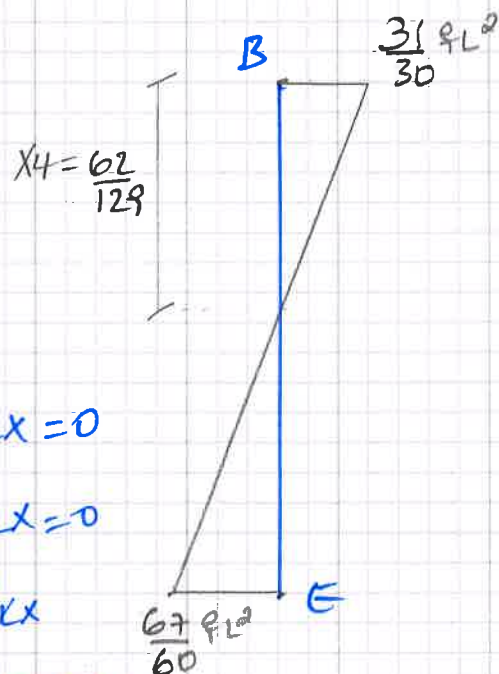


$$M_x = \frac{31}{30} qL^2 - \frac{43}{20} qLX = 0$$

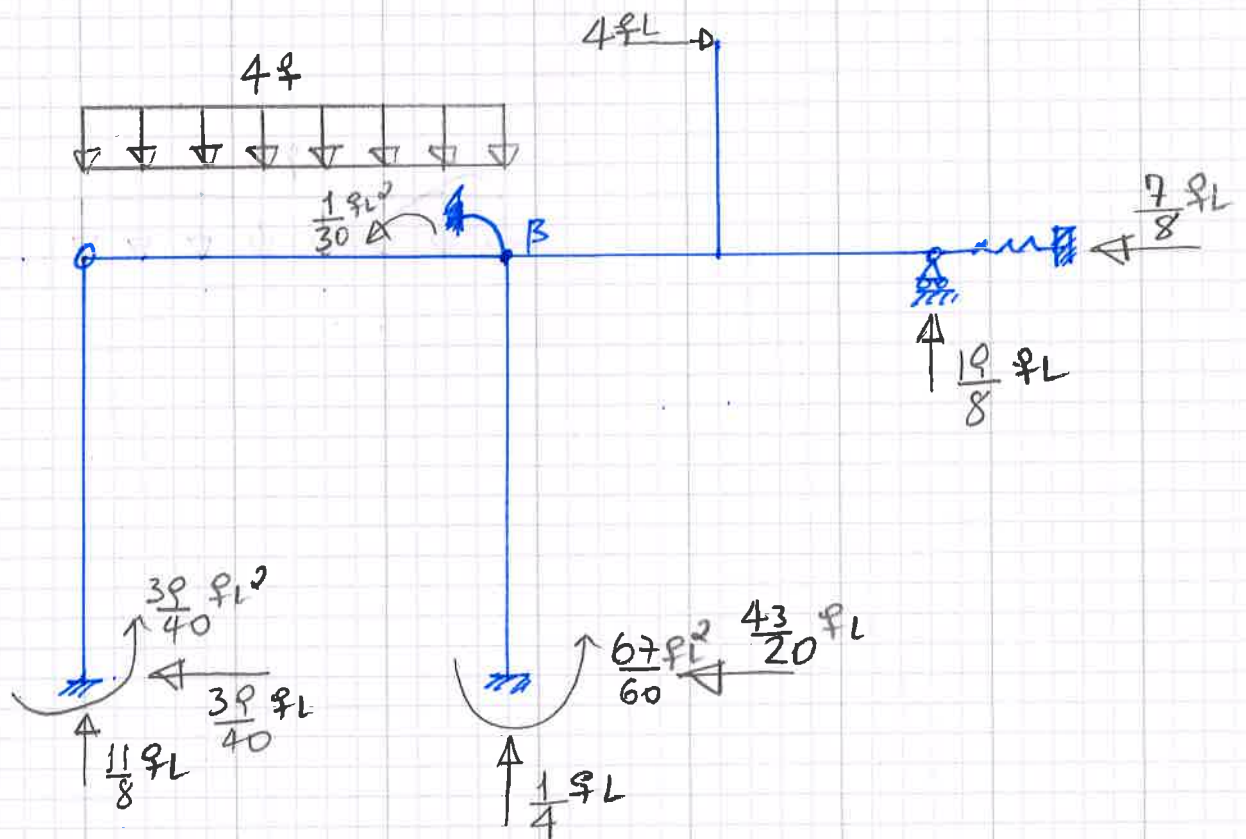
$$\frac{31}{30} qL^2 - \frac{43}{20} qLX = 0$$

$$\frac{31}{30} qL^2 = \frac{43}{20} qLX$$

$$X = \frac{62}{129} L \Rightarrow X_4 = \frac{62}{129} L$$



Global equilibrium



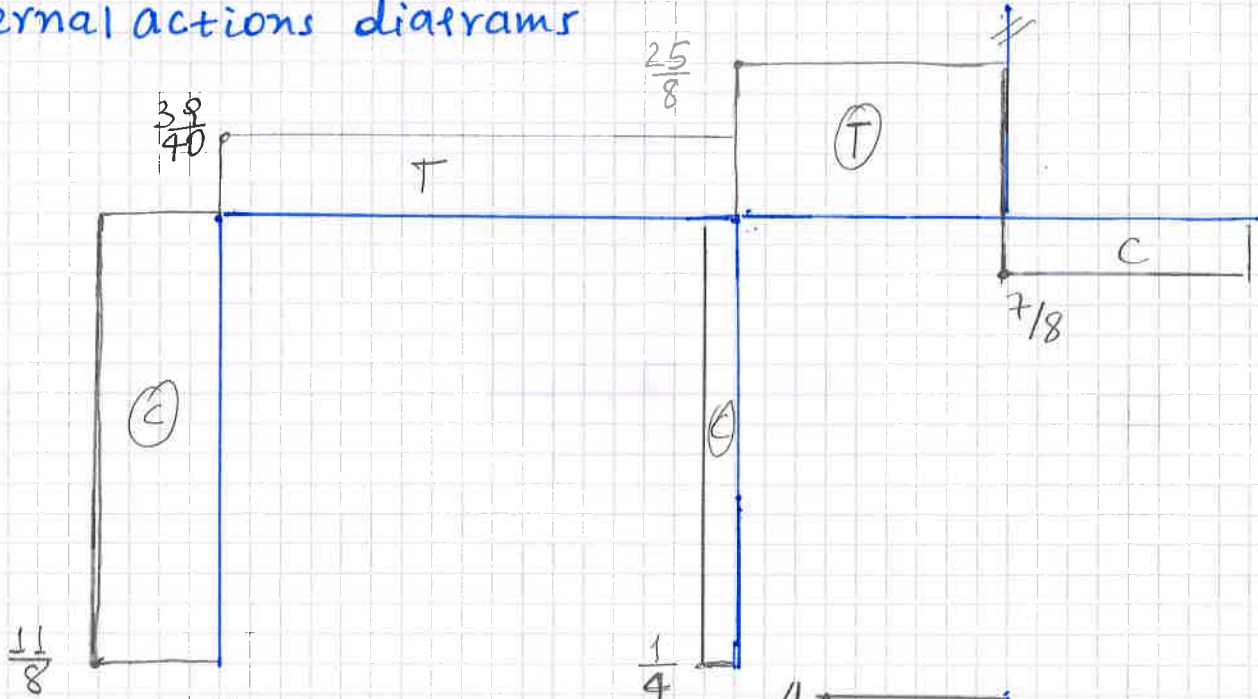
$$\rightarrow \sum F_H = 4qL - \frac{39}{40}qL - \frac{43}{20}qL - \frac{7}{8}qL = 0$$

$$\uparrow \sum F_V = \frac{11}{8}qL + \frac{1}{4}qL + \frac{19}{8}qL - 4qL = 0$$

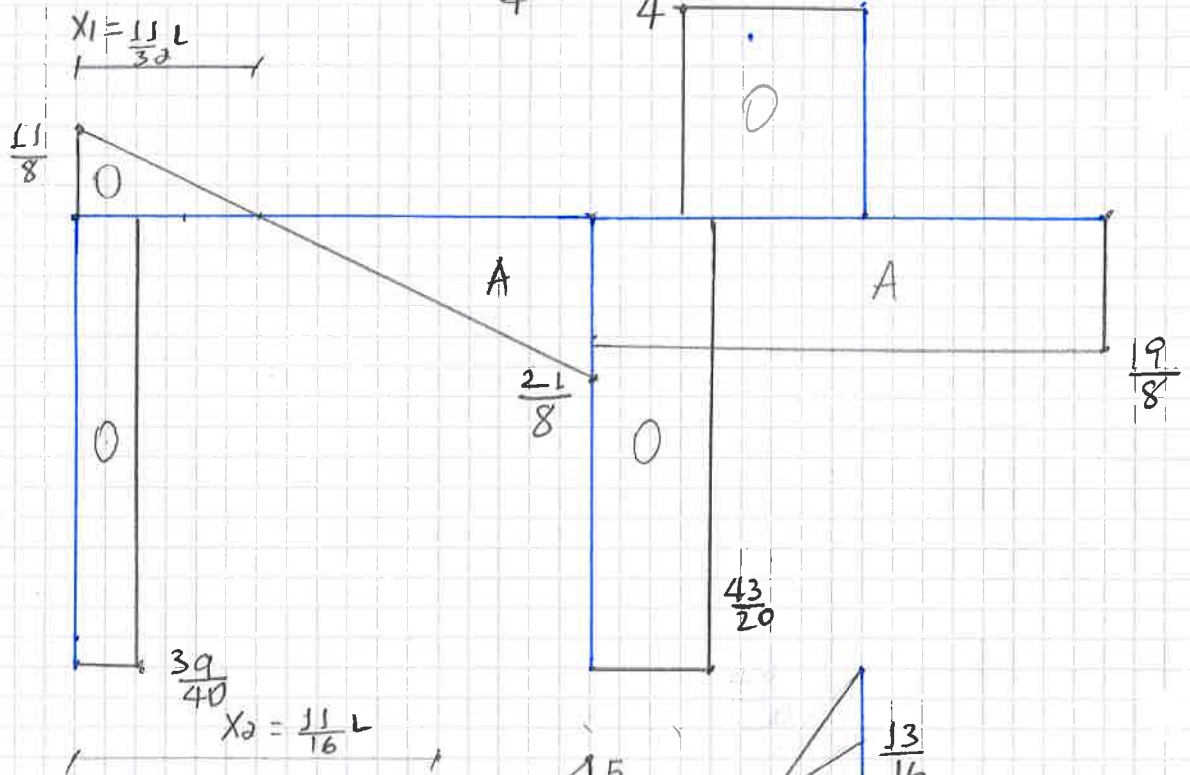
$$\curvearrowleft \sum M_B = \frac{39}{40}qL^2 + \frac{67}{60}qL^2 + \frac{1}{30}qL^2 - 2qL^2 + \frac{19}{8}qL^2 - \frac{43}{20}qL^2 - \frac{11}{8}qL^2 - \frac{39}{40}qL^2 + 2qL^2 = 0$$

Internal actions diagrams

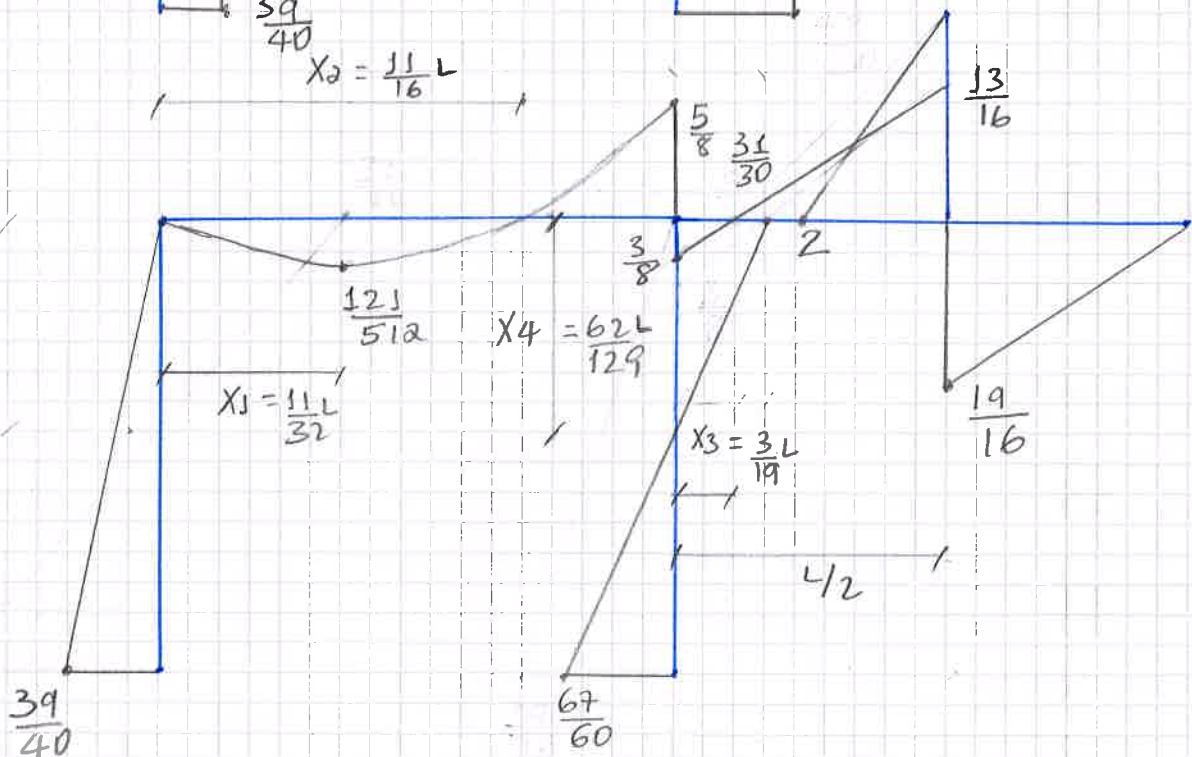
M/q_1



V/q_2



M/q_2



Qualitative Deflected shape

